# Simulating the Effect of Local Environmental Conditions on Human Thermal Comfort

### Dr Paul Cropper De Montfort University

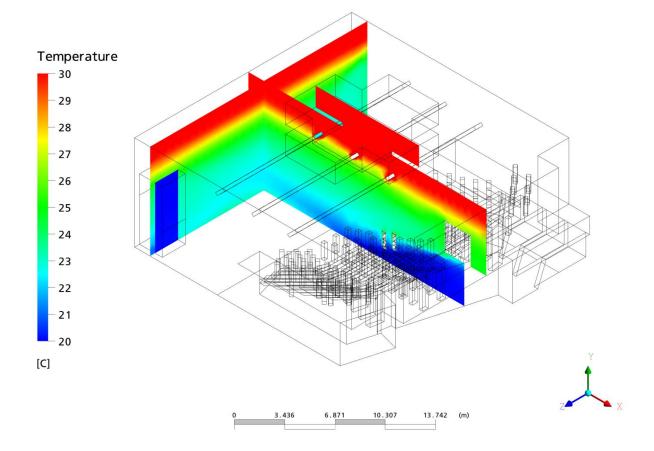




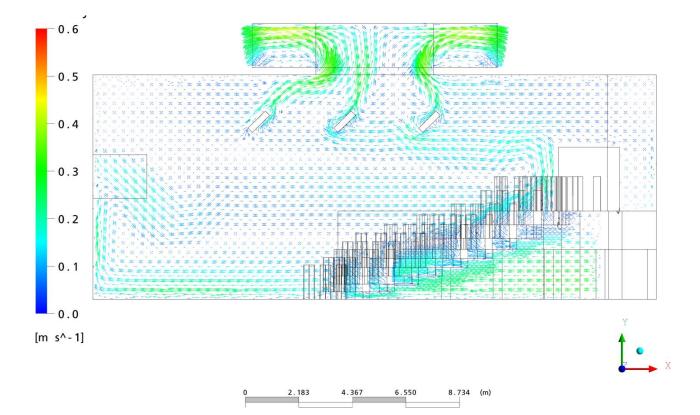
# **Research Project**

- Funded by the UK Engineering and Physical Sciences Research Council (EPSRC)
- Collaborative project between De Montfort University and Loughborough University
- Other collaborators: ANSYS UK

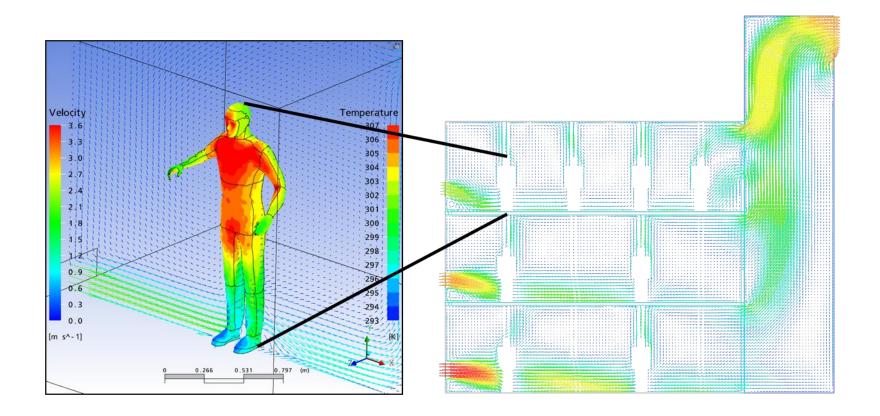
## CFD – temperature distribution



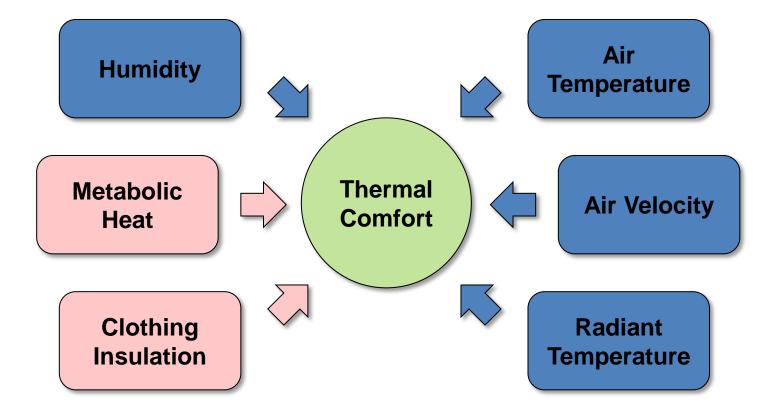
## CFD – airflow distribution



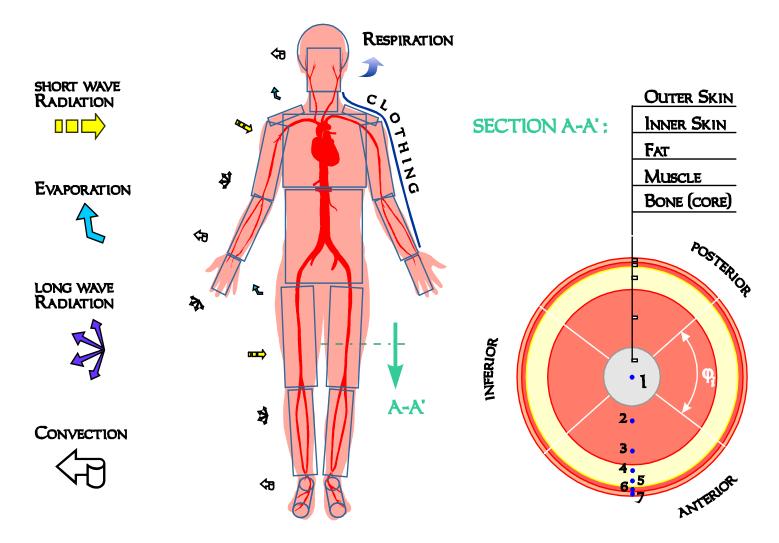
# Modelling a human body in CFD



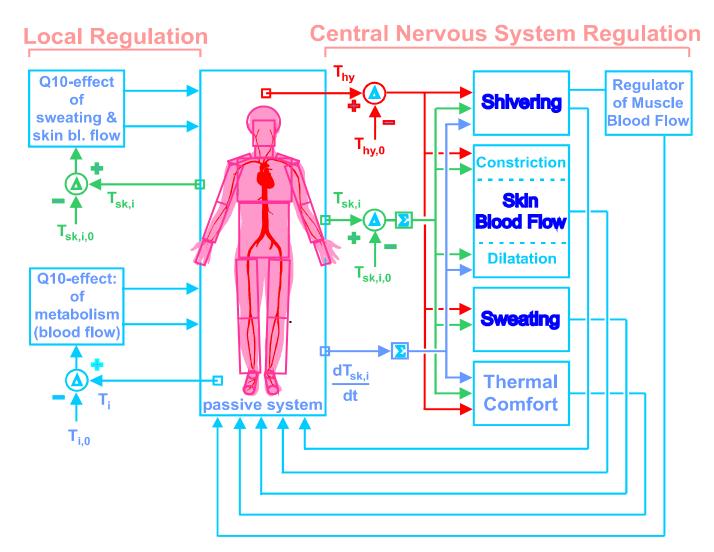
## What is thermal comfort?



# IESD-Fiala model - Passive System

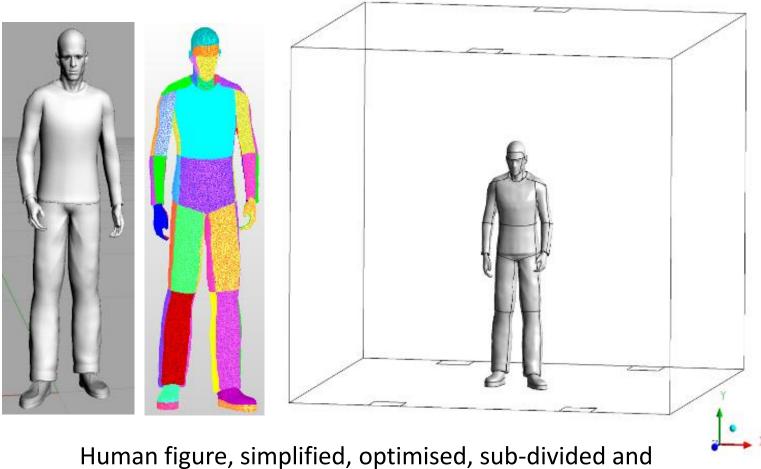


# IESD-Fiala model - Active System



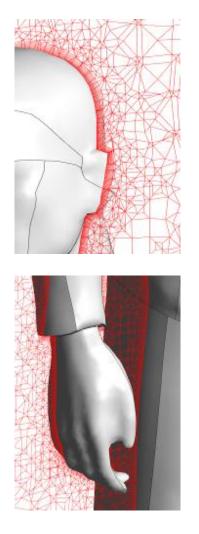
Source: Dr Dusan Fiala

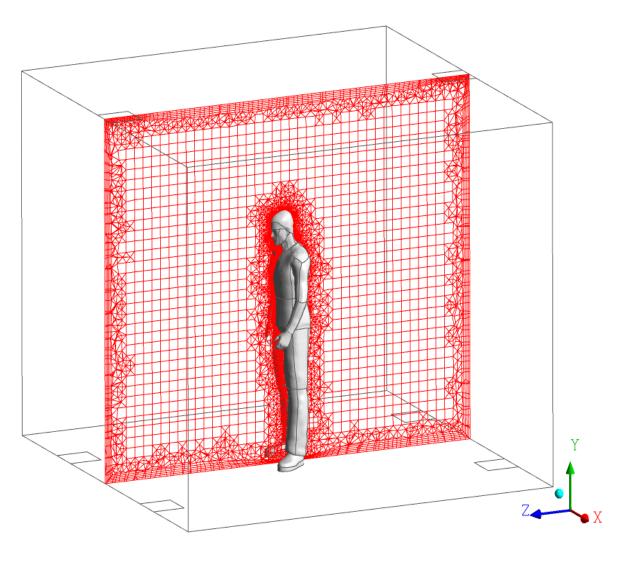
## A human thermal manikin

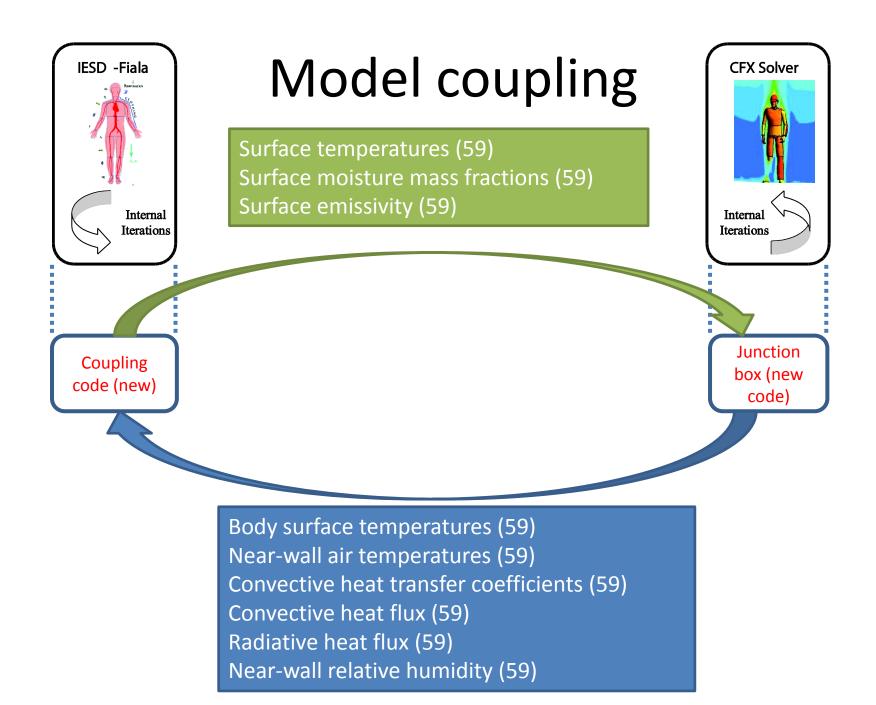


placed in a CFD environment

## The CFD mesh







# Data exchange- CFX

For each patch, read

- 1. Area
- 2. Surface temperature (Tw)
- 3. Convective heat transfer coefficient (CHTC)
- 4. Near-wall water ideal gas mass fraction (Mf)
- 5. Total wall heat flux (Qw)
- 6. Radiative heat flux (Qr)

#### Calculate:

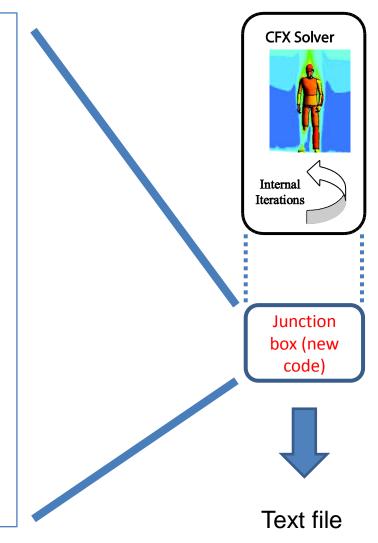
- 1. Qc = Qw Qr
- 2.  $Rh_nw = Psg/Pss \times 100$
- 3. Tnw = Tw Qc/CHTC
- 4.  $Qc = Qc \times Area$
- 5.  $Qr = Qr \times Area$

(area correction for patch area difference)

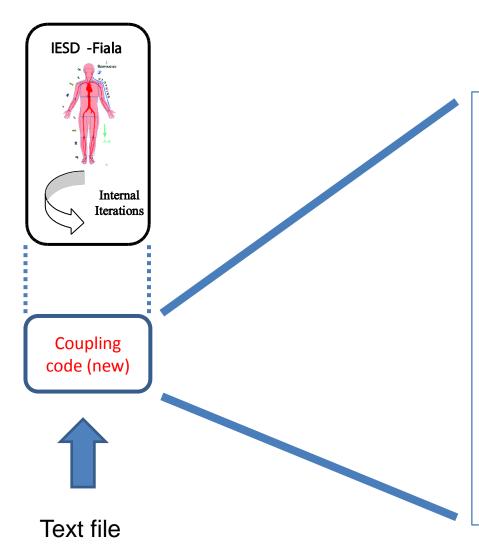
Where:

Pss = Saturated vapour pressure (from Mf and Tnw)

Psg = Vapour pressure of water vapour mixed with dry air (from Tnw)



# Data exchange – IESD-Fiala model



#### Read text file from CFX

#### Calculate:

1. Qc/area x clothing factor

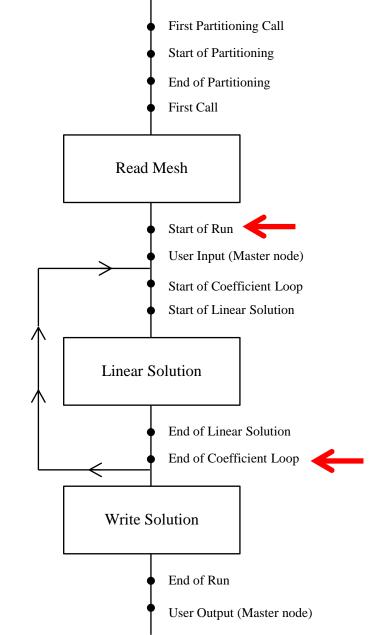
2. Qr/area x clothing factor (area correction for patch area difference and for clothing)

Pass to IESD-Fiala model (for each patch):

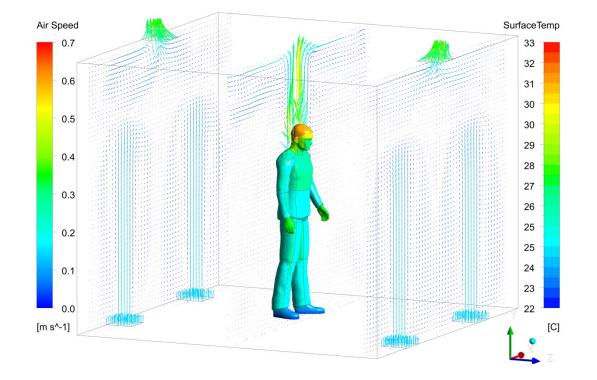
- 1. Near-wall relative humidity
- 2. CHTC
- 3. Tnw
- 4. Qc
- 5. Qr (long wave)

# Solver customisation

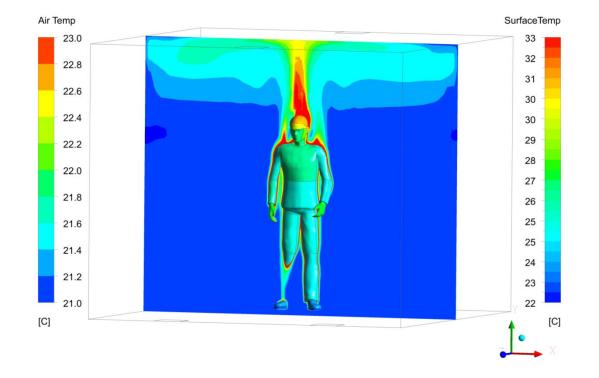
- Chosen CFD solver is ANSYS CFX
- All data is accessed using utility routines
- Junction Box code called at specific points in the execution cycle
- CEL functions replace fixed values



### Air velocity and surface temperature



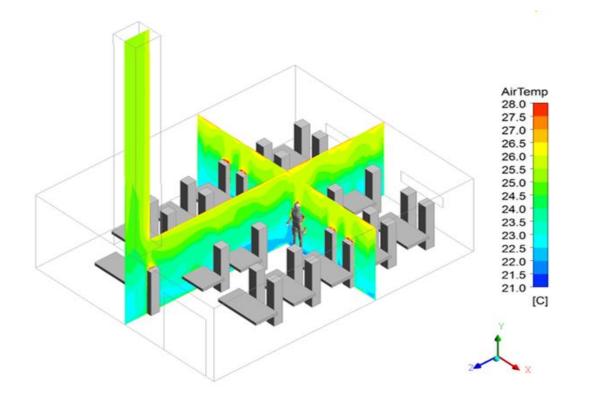
### Air and surface temperatures



# **Example application**



## Air temperature distribution

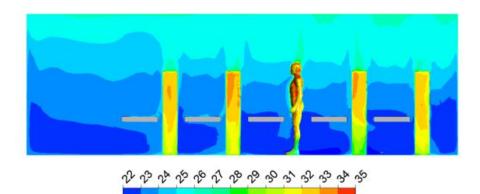


21 °C case

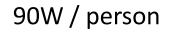
~10 ac/h

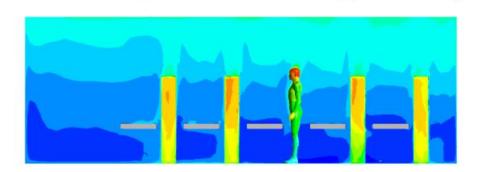
## Body surface and air temperature

[C]



Uncoupled

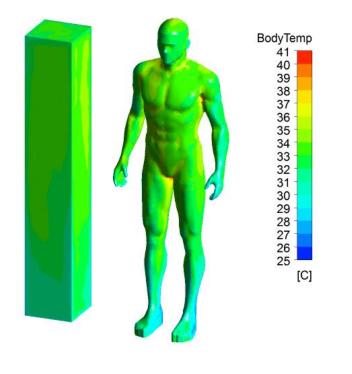




Temp

Coupled

# Influence of ambient temperature

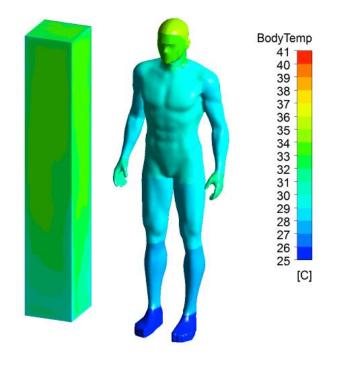


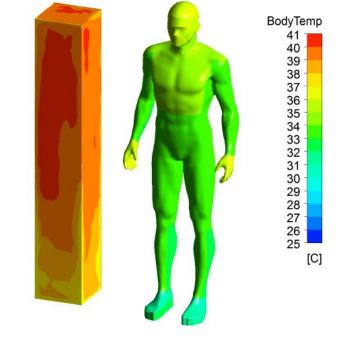
BodyTemp [C]

Uncoupled 28 °C case

Uncoupled 21 °C case

# Influence of ambient temperature



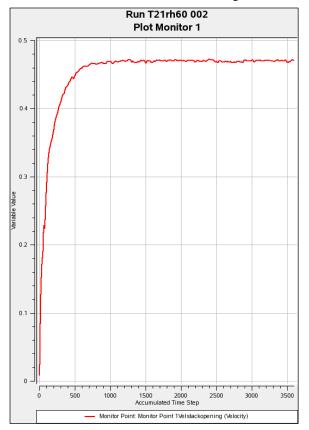


Coupled 28 °C case

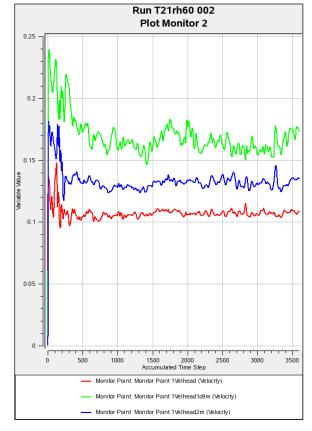
Coupled 21 °C case

## **Transient flow evolution**

**Stack velocity** 

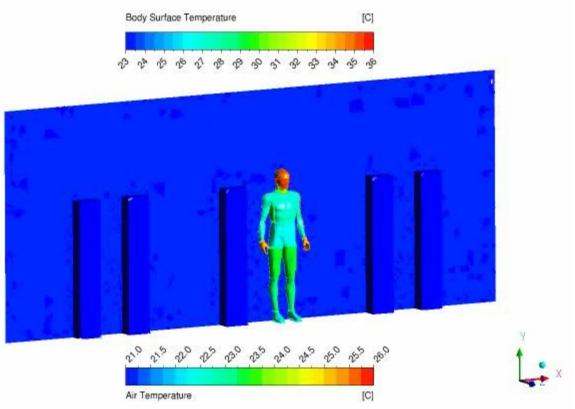




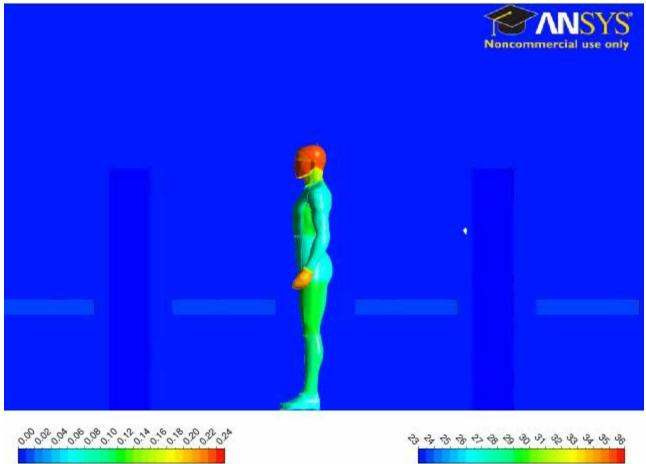


## Transient – temperature evolution





## Transient – velocity evolution



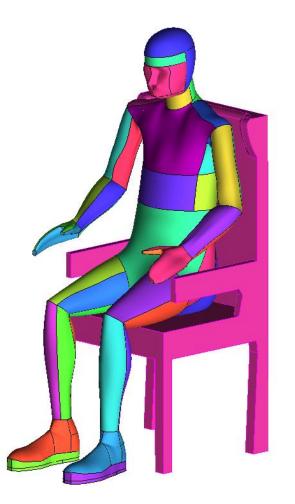
[m s^-1]

Air Speed

Body Surface Temperature

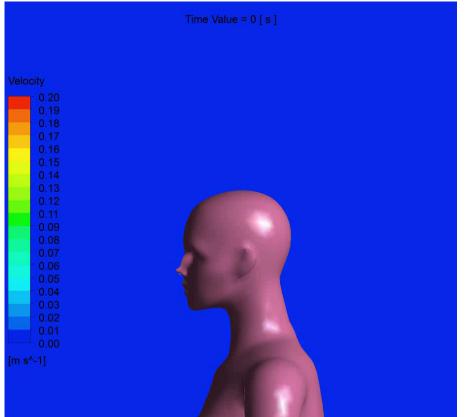
[C]

## Seated thermal manikin



Manikin courtesy of: Sandeep Rao Bolineni Fraunhofer-Institut for Building Physics IBP

# Breathing

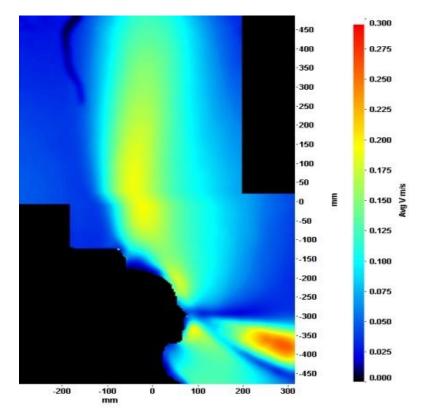


- CO<sub>2</sub> distribution and IAQ (e.g. schools)
- Person to person infection modelling (esp. healthcare buildings)

Source: Loughborough university

## **Empirical validation of breathing**





Source: Loughborough university

# Future Work

- Transient simulation De Montfort University
- Breathing Loughborough University
- New manikins age, height, weight, gender
- New human physiology model
- Moving manikins games technology
- Better clothing models
- Visualisation 3D TV, VR, Holography

## Questions?